# A THEORY OF LONGITUDINALLY POLARISED PIEZOCOMPOSITE ROD BASED ON MINDLIN-HERRMANN MODEL 

M. Shatalov ${ }^{\text {a, }, ~}$, mshatlov@csir.co.za<br>I. Fedotov ${ }^{\text {b }}$, fedotovi@tut.ac.za<br>${ }^{\text {a }}$ Materials Sciences and Manufacturing, Council for science and Industrial research (CSIR)<br>P.O. Box 395, Pretoria 0001, South Africa.<br>${ }^{\mathrm{b}}$ Department of Mathematics and Statistics, Tshwane University of Technology<br>P.O. Box.680, Pretoria, 0001 FIN-40014, South Africa

HM. Tenkam ${ }^{\text {b }}$, DjouosseuTenkamHM@tut.ac.za
${ }^{\mathrm{b}}$ Department of Mathematics and Statistics, Tshwane University of Technology
P.O. Box.680, Pretoria, 0001 FIN-40014, South Africa
T. Fedotova ${ }^{\text {c }}$, Tatiana.Fedotova@wits.ac.za
${ }^{\text {c }}$ School of chemical and metallurgical engineering, University of Witwatersrand
P.O. Box. 3 Wits, Johannesburg, 2050, South Africa


#### Abstract

The conventional theory of the piezoelectric rod is based on an assumption that its lateral vibrations are negligible. In this case the rod, vibration could be described in terms of one-dimensional wave equation and a set of mechanical and electric boundary conditions. However, the main assumption of the model is only valid in the case of long and relatively thin rods. As a rule, the linear dimensions of piezoelectric components used in real transducers are comparable with the characteristic dimensions of their cross-sections, and hence, their lateral displacements have to be taken into consideration. In the present paper, the theory of the piezoelectric rod is developed on the basis of the Mindlin-Herrmann model. In the frame of this theory, the longitudinal and lateral displacements are described by two independent functions and vibration of the rod is obtained in terms of a system of two partial differential equations. The Hamilton variational principle is used for derivation of the system of equations of motion and for obtaining the mechanical and electric boundary conditions. On the basis of the formulated Mindlin-Herrmann model, the electric impedance of the piezoelectric rod is calculated. Possible generalizations of the proposed approach are considered and conclusions are formulated. An example of the application of the piezoelectric rod based on the Mindlin-Herrmann theory is given.


Keywords: lateral displacement, Mindlin-Herrmann model, piezoelectric, thick rod.

## 1. INTRODUCTION

The piezoelectric transducers generate and detect ultrasonic waves in continuous media such as fluids, solids, etc (Risstic, 1983). They have been developed for many industrial applications. The conventional theory of the thickness vibrator transducer is based on an assumption that its lateral vibrations are negligible. In this case, the transducers' dynamics could be described in terms of one-dimensional wave equation and a set of mechanical and electric boundary conditions. However, the main assumption of the model is only valid in the case of long and relatively thin rods. As a rule, the linear dimensions of thickness vibrators are comparable with the characteristic dimensions of their crosssections and hence, it is necessary to take into consideration the lateral displacements of these transducers. Theory of relatively thick transducers was developed (Shatalov et al., 2009b). In the present paper, the theory of the relatively thick piezocomposite rod is developed on the basis of the Mindlin-Herrmann model of longitudinal vibrations of rods. In the frame of this theory, it is supposed that the lateral displacements are proportional to the product of an independent function which is subjected to determination and the distance from the neutral line of the transducers' cross-section (Shatalov et al., 2009a). The Hamilton variational principle is used for derivation of the system of equation of motion and for obtaining the mechanical and electric boundary conditions. On the basis of the obtained Mindlin-Herrmann model, the electric impedance of the thickness vibrator is calculated. Possible generalizations of the proposed approach are considered and conclusions are formulated. General solution of the problem is obtained from the variational principle using two orthoganalities conditions of the eigenfunctions (Fedotov et al., 2009).

## 2. MINDLIN-HERRMANN MODEL FOR THICKNESS VIBRATOR

Suppose that $\mathrm{Oz}(03)$ is the axis of polarization of a piezoelectric rod. According to the Rayleigh-Love and Bishop theories displacements in $O x(01), O y(02), O z$ - directions are correspondingly $u, v$, and $w$ :

$$
\begin{align*}
& u=u(x, z, t)=x \varphi(z, t) \\
& v=v(y, z, t)=y \varphi(z, t)  \tag{1}\\
& w=w(z, t)
\end{align*}
$$

where $\varphi=\varphi(z, t)$ represents the lateral or transverse contraction about the $z$-axis and $x, y$-displacements from the neutral axis of any symmetric cross-section of the bar. Note that the transverse displacements $u$ and $v$ are both connected by the function $\varphi$ and are independent of the longitudinal displacement, $w$. This proves the independence of the shearing deformation thus, the entire motion can be described by two independent functions: $w$ and $\varphi$.

Linear strain are:

Normal strain:

$$
\begin{array}{lll}
S_{1}=\frac{\partial u}{\partial x}=\varphi & S_{4}=\frac{\partial v}{\partial z}+\frac{\partial w}{\partial x}=y \varphi^{\prime} \\
S_{2}=\frac{\partial v}{\partial y}=\varphi & \text { and shear strain: } & S_{5}=\frac{\partial u}{\partial z}+\frac{\partial w}{\partial y}=x \varphi^{\prime} \\
S_{3}=\frac{\partial w}{\partial x}=w^{\prime} & S_{6}=0
\end{array}
$$

Linear stress-strain relations are:

$$
\begin{array}{lll} 
& T_{1}=c_{11}^{D} S_{1}+c_{12}^{D} S_{2}+c_{13}^{D} S_{3}-h_{31} D_{3} & T_{4}=c_{44}^{D} S_{4} \\
\text { Normal stress: } & T_{2}=c_{12}^{D} S_{1}+c_{11}^{D} S_{2}+c_{13}^{D} S_{3}-h_{31} D_{3}  \tag{2}\\
& T_{3}=c_{13}^{D}\left(S_{1}+S_{2}\right)+c_{33}^{D} S_{3}-h_{33} D_{3} & \text { and shear stress: } \\
& T_{5}=c_{44}^{D} S_{5} \\
& T_{6}=0
\end{array}
$$

Where $c_{i j}^{D}$ - elastic constants at constant electric charge density and $D_{1}=D_{2}=0, D_{3}=D_{30} e^{i \omega t}$ are the electric displacements in which $D_{30}=$ const and $\omega=2 \pi f$ - frequency of excitation.

Expression for kinetic energy:

$$
\begin{equation*}
T=\frac{\rho}{2} \int_{0}^{1}\left[A(z) \dot{w}^{2}+I_{p} \dot{\varphi}^{2}\right] d z \tag{3}
\end{equation*}
$$

where $l$ - thickness of the transducer, $A=A(z)$ is the cross-section area and $I_{p}=\int_{(A)}\left(x^{2}+y^{2}\right) d A$ is the polar moment of inertia

Strain energy is as follows:

$$
\begin{align*}
P & =\frac{1}{2} \int_{0}^{l} \int_{(A)}\left(T_{1} S_{1}+T_{2} S_{2}+T_{3} S_{3}+T_{4} S_{4}+T_{5} S_{5}+T_{6} S_{6}\right) d A d z \\
& =\frac{1}{2} \int_{0}^{l}\left\{\left[2\left(c_{11}^{D}+c_{12}^{D}\right) \varphi^{2}+c_{33}^{D} w^{\prime 2}+4 c_{13}^{D} w^{\prime} \varphi\right] A+I_{p} c_{44}^{D} \varphi^{\prime 2}-\left(2 h_{31} \varphi+h_{33} w^{\prime}\right) A D_{3}\right\} d z \tag{4}
\end{align*}
$$

where $h_{i j}$ is the piezoelectric coefficient and $A=\int_{(A)} d A$
Electric energy is:

$$
\begin{align*}
W & =\frac{1}{2} \int_{0}^{1} \int_{(A)}\left(E_{1} D_{1}+E_{2} D_{2}+E_{3} D_{3}\right) d A d z \\
& =\int_{0}^{1} \int_{(A)}\left\{-h_{31} S_{1} D_{3}-\frac{1}{2} h_{33} S_{3} D_{3}+\frac{1}{2} \beta_{3}^{S} D_{3}^{2}\right\} d A d z  \tag{5}\\
& =\frac{1}{2} \int_{0}^{l}\left(\beta_{3}^{S} A D_{3}^{2}-2 h_{31} A D_{3} \varphi-h_{33} A D_{3} w^{\prime}\right) d z
\end{align*}
$$

where $\beta_{3}^{s}$ is the dielectric constant
It is also necessary to keep in mind the following electric boundary condition:

$$
\begin{equation*}
\int_{0}^{l} E_{3} d z=V=V(t) \tag{6}
\end{equation*}
$$

where $V(t)=V_{0} e^{i \omega t} \quad\left(V_{0}=\right.$ const $)$ - excitation voltage applied at the edge of the thickness vibrator and $E_{3}=-2 h_{31} \varphi-h_{33} w^{\prime}+\beta_{3}^{s} D_{3}$ is the electric field on the $Z$-axis.

Total Lagrangian of the system:

$$
\begin{align*}
L & =K-P+W+\lambda \cdot\left[\int_{0}^{l} E_{3} d z-V(t)\right] \\
& =\frac{1}{2} \int_{0}^{l}\left\{\left[A \dot{w}^{2}+I_{p} \dot{\varphi}^{2}\right] \rho-\left[2\left(c_{11}^{D}+c_{12}^{D}\right) \varphi^{2}+c_{33}^{D} w^{\prime 2}+4 c_{13}^{D} w^{\prime} \varphi\right] A-I_{p} c_{44}^{D} \varphi^{\prime 2}+\beta_{3}^{s} A D_{3}^{2}\right\} d z-2 \lambda h_{31} \int_{0}^{l} \varphi d z-\lambda h_{33}(w(l)-w(0))+\beta_{3}^{s} \lambda l D_{3} \tag{7}
\end{align*}
$$

where $\lambda$ is the Lagrange multiplier
We can rewrite the pervious expression in the compact form as follow:

$$
\begin{equation*}
L=L\left(\dot{w}, w^{\prime}, \varphi, \dot{\varphi}, \varphi^{\prime}, D_{3}, \lambda\right)=\int_{0}^{l} \Lambda\left(\dot{w}, w^{\prime}, \varphi, \dot{\varphi}, \varphi^{\prime}, D_{3}, \lambda\right) d z+\Lambda_{0}\left(D_{3}, \lambda, w(l), w(0)\right) \tag{7a}
\end{equation*}
$$

where

$$
\begin{align*}
\Lambda & =\Lambda\left(\dot{w}, w^{\prime}, \varphi, \dot{\varphi}, \varphi^{\prime}, D_{3}, \lambda\right) \\
& =\frac{1}{2}\left\{\left[A \dot{w}^{2}+I_{p} \dot{\varphi}^{2}\right] \rho-\left[2\left(c_{11}^{D}+c_{12}^{D}\right) \varphi^{2}+c_{33}^{D} w^{\prime 2}+4 c_{13}^{D} w^{\prime} \varphi\right] A-I_{p} c_{44}^{D} \varphi^{\prime 2}+\beta_{3}^{S} A D_{3}^{2}-4 \lambda h_{31} \varphi\right\} \tag{8}
\end{align*}
$$

is the Lagrange density and

$$
\begin{equation*}
\Lambda_{0}=\Lambda_{0}\left(D_{3}, \lambda, w(l), w(0)\right)=\beta_{3}^{s} \lambda l D_{3}-\lambda h_{33}(w(l)-w(0)) \tag{9}
\end{equation*}
$$

Applying the Hamiltonian variational principle to Lagrangian (7a) we obtain the system of equation of motion in the compact form:

$$
\left\{\begin{array}{l}
\frac{\partial}{\partial t}\left(\frac{\partial \Lambda}{\partial \dot{w}}\right)+\frac{\partial}{\partial z}\left(\frac{\partial \Lambda}{\partial w^{\prime}}\right)=0 \\
\frac{\partial}{\partial t}\left(\frac{\partial \Lambda}{\partial \dot{\varphi}}\right)+\frac{\partial}{\partial z}\left(\frac{\partial \Lambda}{\partial \varphi^{\prime}}\right)-\frac{\partial \Lambda}{\partial \varphi}=0
\end{array}\right.
$$

with the mechanical boundary conditions

$$
\begin{gathered}
-\left.\frac{\partial \Lambda}{\partial w^{\prime}}\right|_{x=0}+\frac{\partial \Lambda_{0}}{\partial w(0)}=0 \\
\left.\frac{\partial \Lambda}{\partial w^{\prime}}\right|_{x=l}+\frac{\partial \Lambda_{0}}{\partial w(l)}=0
\end{gathered}
$$

$$
\text { or }\left.\quad w\right|_{x=0, l}=0
$$

and

$$
\left.\varphi^{\prime}\right|_{x=0, l}=0 \quad \text { or }\left.\quad \varphi\right|_{x=0, l}=0
$$

and the electrical boundary conditions:

$$
\begin{aligned}
& \frac{\partial \Lambda}{\partial D_{3}}+\frac{\partial \Lambda_{0}}{\partial D_{3}}=0 \\
& \frac{\partial \Lambda}{\partial \lambda}+\frac{\partial \Lambda_{0}}{\partial \lambda}=0
\end{aligned}
$$

The explicit form of the Mindlin-Herrmann model for the polarised piezocomposite thick rod is given as follow.

$$
\left\{\begin{array}{l}
\rho A \frac{\partial^{2} w}{\partial t^{2}}-c_{33}^{D} A \frac{\partial^{2} w}{\partial z^{2}}-2 c_{13}^{D} A \frac{\partial \varphi}{\partial z}=0  \tag{10}\\
\rho I_{p} \frac{\partial^{2} \varphi}{\partial t^{2}}-c_{44}^{D} I_{p} \frac{\partial^{2} \varphi}{\partial z^{2}}+2 A\left(c_{11}^{D}+c_{12}^{D}\right) \varphi+2 c_{13}^{D} A \frac{\partial w}{\partial z}+2 \lambda h_{31}=0
\end{array}\right.
$$

associated boundary conditions

$$
\begin{equation*}
c_{33}^{D} A w^{\prime}+\left.2 c_{13}^{D} A \varphi\right|_{x=0, l}+\lambda h_{33}=0 \tag{11}
\end{equation*}
$$

or $\left.\quad w\right|_{x=0, l}=0$
and

$$
\begin{equation*}
\left.\varphi^{\prime}\right|_{x=0, l}=0 \tag{13}
\end{equation*}
$$

$$
\text { or }\left.\quad \varphi\right|_{x=0, l}=0
$$

and the electric boundary condition

$$
\begin{align*}
& A D_{3}+\lambda l=0  \tag{15}\\
& 2 h_{31} \varphi+h_{33}(w(l)-w(0))-\beta_{3}^{s} l D_{3}=0 \tag{16}
\end{align*}
$$

From equation (15), we have $\lambda=-\frac{A D_{3}}{l}$, and substituting it into (10) and (11), this leads to:

$$
\left\{\begin{array}{l}
\rho A \frac{\partial^{2} w}{\partial t^{2}}-c_{33}^{D} A \frac{\partial^{2} w}{\partial z^{2}}-2 c_{13}^{D} A \frac{\partial \varphi}{\partial z}=0  \tag{17}\\
\rho I_{p} \frac{\partial^{2} \varphi}{\partial t^{2}}-c_{44}^{D} I_{p} \frac{\partial^{2} \varphi}{\partial z^{2}}+2 A\left(c_{11}^{D}+c_{12}^{D}\right) \varphi+2 c_{13}^{D} A \frac{\partial w}{\partial z}=\frac{2}{l} A h_{31} D_{3}
\end{array}\right.
$$

with the electro-mechanic boundary conditions (here we choose these boundary conditions without loosing the generality stated in our main objective):

$$
\begin{equation*}
c_{33}^{D} A w^{\prime}+\left.2 c_{13}^{D} A \varphi\right|_{x=0, l}=\frac{h_{33}}{l} D_{3} \quad \text { (18) } \quad \text { and }\left.\quad \varphi\right|_{x=0, l}=0 \tag{19}
\end{equation*}
$$

The main problem is entirely defined by considering the initial conditions:

$$
\begin{equation*}
\left.w(z, t)\right|_{t=0}=g(z),\left.\dot{w}(z, t)\right|_{t=0}=h(z) \quad \text { and }\left.\quad \varphi(z, t)\right|_{t=0}=\phi(z),\left.\dot{\varphi}(z, t)\right|_{t=0}=q(z) \tag{20}
\end{equation*}
$$

## 3. FREE VIBRATION PROBLEM

In this section, we are dealing with the free vibration problem, that means $D_{3}=0$ (no electric displacement). Thus we apply the Fourier method to the obtained homogeneous problem. Let us assume that $w(z, t)=Z(z) e^{i \omega t}$ and $\varphi(z, t)=\Phi(z) e^{i \omega t}$ where $i^{2}$ and $\omega$ is the circular frequency. This leads to the Sturm-Liouville problem:

$$
\left\{\begin{array}{l}
c_{33} Z^{\prime \prime}+2 c_{13} \Phi^{\prime}=-\omega^{2} \rho Z  \tag{21}\\
c_{44} I_{p} \Phi^{\prime \prime}-2 A\left(c_{11}+c_{12}\right) \Phi-2 c_{13} A Z^{\prime}=-\omega^{2} \rho I_{p} \Phi
\end{array}\right.
$$

With the associated boundary conditions

$$
\begin{equation*}
c_{33} Z^{\prime}+\left.2 c_{13} \Phi\right|_{z=0, l}=0 \text { and }\left.\Phi\right|_{z=0, l}=0 \tag{22}
\end{equation*}
$$

The above Sturm-liouville problem is unusual (two-dimension). So, to compliment the lack of theory in this particular problem, we trade the difficulty by considering the case of a travelling wave. Thus we can write:

$$
\begin{equation*}
Z(z)=Z_{0} e^{-i k z} \text { and } \Phi(z)=\Phi_{0} e^{-i k z} \tag{23}
\end{equation*}
$$

where $Z_{0}$ and $\Phi_{0}$ are respectively the longitudinal and lateral non zero amplitude (unknowns) and $k$ is the wave number and need to be determined.
Hence system (21) becomes:

$$
\left\{\begin{array}{l}
\left(\omega^{2} \rho-c_{33} k^{2}\right) Z_{0}-2 i k c_{13} \Phi_{0}=0  \tag{24}\\
2 i k A c_{13} Z_{0}+\left[\omega^{2} \rho I_{p}-c_{44} I_{p} k^{2}-2 A\left(c_{11}+c_{12}\right)\right] \Phi_{0}=0
\end{array}\right.
$$

Solving the characteristic equation of the determinant of system (21) for $k$, we obtain:
$k_{1,2}^{2}=k_{1,2}^{2}(\omega)=\frac{\beta \pm \sqrt{\beta^{2}+4 \alpha \gamma}}{2 \alpha}$
where $\alpha=c_{33}^{D} c_{44}^{D} I_{p}, \beta=\beta(\omega)=\left[2 A\left(c_{11}^{D}+c_{12}^{D}\right) c_{33}^{D}-4\left(c_{13}^{D}\right)^{2}-\omega^{2} \rho\left(c_{44}^{D}+I_{p} c_{33}^{D}\right)\right]$ and $\gamma=\gamma(\omega)=\omega^{2} \rho\left[\omega^{2} \rho I_{p}-2 A\left(c_{11}^{D}+c_{12}^{D}\right)\right]$

Using formula (23) we can express $Z(z)$ as follows:

$$
Z=Z(z, \omega)=a_{1} Z_{1}\left(k_{1} z\right)+a_{2} Z_{2}\left(k_{1} z\right)+a_{3} Z_{3}\left(k_{2} z\right)+a_{4} Z_{4}\left(k_{2} z\right)
$$

where $Z_{1,3}\left(k_{j} z\right)=\left\{\begin{array}{l}\cos \left(k_{j} z\right) \text { if } \operatorname{Im}\left[k_{j}\right]=0 \\ \cosh \left(k_{j} z\right) \text { orthewise }\end{array}\right.$ and $Z_{2,4}\left(k_{j} z\right)=\left\{\begin{array}{l}\sin \left(k_{j} z\right) \text { if } \operatorname{Im}\left[k_{j}\right]=0 \\ \sinh \left(k_{j} z\right) \text { orthewise }\end{array}(j=1,2)\right.$
and the constants $a_{1}, a_{2}, a_{3}$ and $a_{4}$ are not all equal to zero.
Without loss the generality, the case $\operatorname{Im}\left[k_{1}\right]=0$ is considered in the discussion that follows. Hence

$$
\begin{equation*}
Z=Z(z, \omega)=a_{1} \cos \left(k_{1} z\right)+a_{2} \sin \left(k_{1} z\right)+a_{3} \cosh \left(k_{2} z\right)+a_{4} \sinh \left(k_{2} z\right) \tag{25}
\end{equation*}
$$

Substituting (25) into the first equation of system (21) and solving the obtained equation for $\Phi$ :

$$
\begin{align*}
\Phi=\Phi(z, \omega)= & -\frac{1}{2 c_{13}^{D}}\left[a_{1}\left(\frac{\omega^{2} \rho}{k_{1}}-k_{1} c_{33}^{D}\right) \sin \left(k_{1} z\right)-a_{2}\left(\frac{\omega^{2} \rho}{k_{1}}-k_{1} c_{33}^{D}\right) \cos \left(k_{1} z\right)\right]- \\
& -\frac{1}{2 c_{13}^{D}}\left[+a_{3}\left(\frac{\omega^{2} \rho}{k_{2}}+k_{2} c_{33}^{D}\right) \sinh \left(k_{2} z\right)+a_{4}\left(\frac{\omega^{2} \rho}{k_{2}}+k_{2} c_{33}^{D}\right) \cosh \left(k_{2} z\right)\right] \tag{26}
\end{align*}
$$

Substituting $\mathrm{Eq}(25)$ and $\mathrm{Eq}(26)$ into the boundary conditions (22), yields a system of four equations with four unknowns: $a_{1}, a_{2}, a_{3}$ and $a_{4}$. Solving the characteristic equation of the determinant for $\omega$ using the method developed by Fedotov et al. (2008) for solving transcendental equation, we obtain many positive roots $\omega_{n}, n=1,2, \ldots$, so called eigenvalues corresponding to the couple of eigenfunction $\left(Z_{n}\left(z, \omega_{n}\right), \Phi_{n}\left(z, \omega_{n}\right)\right)$.

## 4. SOLUTION OF THE FORCED VIBRATION PROBLEM

### 4.1. Orthogonalities of the eigenfunctions

Using system (21) and boundary conditions (22) we can prove two kind of orthogonality condition of the eigenfunctions:

First orthogonality

$$
\left\|\left(z_{n}, \Phi_{n}\right),\left(z_{m}, \Phi_{m}\right)\right\|_{1}=\int_{0}^{l}\left\{I_{p} Z_{n} Z_{m}+A \Phi_{n} \Phi_{m}\right\} d z=\left\|\left(z_{n}, \Phi_{n}\right)\right\|_{1}^{2} \delta_{n m}
$$

where $\delta_{n m}$ is the Kroneker's symbol and $\left\|\left(z_{n}, \Phi_{n}\right)\right\|_{1}^{2}=\int_{0}^{1}\left\{I_{p} Z_{n}^{2}+A \Phi_{n}^{2}\right\} d z$ is the associated square norm Second orthogonality

$$
\left\|\left(Z_{n}, \Phi_{n}\right),\left(Z_{m}, \Phi_{m}\right)\right\|_{2}=\int_{0}^{l}\left\{A c_{33}^{D} Z_{n}^{\prime} Z_{m}^{\prime}+2 A\left(c_{11}^{D}+c_{12}^{D}\right) \Phi_{n} \Phi_{m}+c_{44}^{D} I_{p} \Phi_{n}^{\prime} \Phi_{m}^{\prime}+2 c_{13}^{D} A\left(Z_{n}^{\prime} \Phi_{m}+Z_{m}^{\prime} \Phi_{n}\right)\right\} d=\left\|\left(Z_{n}, \Phi_{n}\right)\right\|_{2}^{2} \delta_{n m}
$$

Where $\left\|\left(Z_{n}, \Phi_{n}\right)\right\|_{2}^{2}=\int_{0}^{1}\left\{A c_{33}^{D} Z_{n}^{\prime 2}+2 A\left(c_{11}^{D}+c_{12}^{D}\right) \Phi_{n}^{2}+c_{44}^{D} I_{p} \Phi_{n}^{\prime 2}+4 c_{13}^{D} A Z_{n}^{\prime} \Phi_{n}\right\} d z$ is the associated square norm.

### 4.2. Solution of the problem

In this section, we give the solution of problem (17)-(20) on the basis of the method of eigenfunction orthogonalities for vibration problem developed by Djouosseu (2008) and Fedotov et al. (2009).

$$
\begin{aligned}
w(z, t) & =A \int_{0}^{l}\left\{g(\xi) \frac{\partial G_{1}(z, \xi, t)}{\partial t}+h(\xi) G_{1}(z, \xi, t)\right\} d \xi+I_{p} \int_{0}^{l}\left\{\phi(\xi) \frac{\partial G_{2}(z, \xi, t)}{\partial t}+q(\xi) G_{2}(z, \xi, t)\right\} d \xi+ \\
& +\frac{A}{\rho} \int_{0}^{1} \int_{0}^{l} F(\tau) G_{1}(z, \xi, t-\tau) d \xi d \tau \\
\varphi(z, t) & =A \int_{0}^{l}\left\{g(\xi) \frac{\partial G_{3}(z, \xi, t)}{\partial t}+h(\xi) G_{3}(z, \xi, t)\right\} d \xi+I_{p} \int_{0}^{l}\left\{\phi(\xi) \frac{\partial G_{4}(z, \xi, t)}{\partial t}+q(\xi) G_{4}(z, \xi, t)\right\} d \xi+ \\
& +\frac{A}{\rho} \int_{0}^{l} \int_{0}^{1} F(\tau) G_{3}(z, \xi, t-\tau) d \xi d \tau
\end{aligned}
$$

where $G_{1}(z, \xi, t)=\sum_{n=1}^{\infty}\left(\frac{Z_{n}(z) Z_{n}(\xi) \sin \Omega_{n} t}{\Omega_{n}\left\|\left(Z_{n}, \Phi_{n}\right)\right\|_{1}^{2}}\right), G_{2}(z, \xi, t)=\sum_{n=1}^{\infty}\left(\frac{Z_{n}(z) \Phi_{n}(\xi) \sin \Omega_{n} t}{\Omega_{n}\left\|\left(Z_{n}, \Phi_{n}\right)\right\|_{1}^{2}}\right)$,
$G_{3}(z, \xi, t)=\sum_{n=1}^{\infty}\left(\frac{\Phi_{n}(z) Z_{n}(\xi) \sin \Omega_{n} t}{\Omega_{n}\left\|\left(Z_{n}, \Phi_{n}\right)\right\|_{1}^{2}}\right)$, and $G_{4}(z, \xi, t)=\sum_{n=1}^{\infty}\left(\frac{\Phi_{n}(z) \Phi_{n}(\xi) \sin \Omega_{n} t}{\Omega_{n}\left\|\left(Z_{n}, \Phi_{n}\right)\right\|_{1}^{2}}\right)$ are Green's function in which
$\Omega_{n}=\frac{1}{\rho} \frac{\left\|\left(Z_{n}, \Phi_{n}\right)\right\|_{2}}{\left\|\left(Z_{n}, \Phi_{n}\right)\right\|_{1}}$ is the natural frequency and $F(t)=\frac{2}{l} A h_{31} D_{3}=\frac{2}{l} A h_{31} D_{30} e^{\omega t}$.

### 4.3. Electrical response of the Rod

The electrical response of the rod to the excitation voltage $V(t)$ applied at the edge of its thickness is characterised by:

- the current through this thickness

$$
I(\omega)=A \dot{D}_{3}=i \omega A\left|D_{3}\right| \text { and }
$$

- the associated electric impedance

$$
Z_{i m p}(\omega)=\frac{V(t)}{I(\omega)}
$$

## 5. CONCLUSIONS

1. The Mindlin-Herrmann approach was used to build a model describing a thick and short piezocomposite rod longitudinally polarised.
2. The Hamilton variational principle was used to derive the system of equation of motion in the process of which the electromechanical boundary conditions were obtained. The method of eigenfunction orthogonalies based on the variational principle was used to obtain the exact solution of the problem in terms of the Green function.
3. The electric impedance through the thickness of the piezocomposite rod is formulated in terms of the excitation frequency.

## 6. REFERENCES

Djouosseu Tenkam, H.M., 2008. "Application of Hyperbolic Equations to Vibration Theories", Master Dissertation, Tshwane University of Technology (Pretoria), South Africa.
Fedotov, I., Shatalov, M., Tenkam, H.M. and Fedotova, T., July-2009. "Application of Eigenfunction Orthogonalities to Vibration Problems", Accepted for publication in the Proceedings of the World Congress on Engineering, London, United Kingdom
Fedotov, I., Shatalov, M. and Mwambakana, J.N., 2008. "Roots of Transcendental Algebraic Equations: A Method of Bracketing Roots and Selecting Initial Estimations", Proceeding of the international conference TIME-2008, Buffletspoort, South Africa.
Rissitic, V.M., 1983. "Principle of Acoustic Devices", John Willey and Sons, New York.
Shatalov, M., Fedotov, I., Tenkam, H.M. and Fedotova, T., October-2009b. "A Theory of Piezoelectric Thickness Vibrator Transducer Based on the Rayleigh and Bishop Model", Submitted for publication in the proceedings of the International Conference on Mathematical Science and Engineering, Venice, Italy.
Shatalov, M., Fedotov, I., Tenkam, H.M. and Marais, J.P., July-2009a. "Comparison of Classical and Modern Theories of Longitudinal Wave Propagation in Elastic Rods", Accepted for publication in the Proceedings of the $16^{\text {th }}$ International Congress on Sound and Vibration, Krakow, Poland.

## 7. RESPONSIBILITY NOTICE

The authors are the only responsible for the printed material included in this paper.

